### REMARKS

In view of the above amendments and following remarks, reconsideration and further examination are requested.

The specification and abstract have been reviewed and revised to make editorial changes thereto and generally improve the form thereof, and a substitute specification and abstract are provided. No new matter has been added by the substitute specification and abstract.

By the current Amendment claims 1-5 have been cancelled and claims 6-25 have been added.

The instant invention pertains to an electrically-operated steering lock device that is prevented from locking a steering shaft even if an electrically-operated member has malfunctioned. With reference to Figures 2, 6A, 9A and 10A, for example, the inventive steering lock device comprises a lock bolt 1 that is movable between a protruding position at which a steering shaft 97 is locked and a retreat position at which the steering shaft is unlocked, and a cam member 6 that is rotatable by an electric motor 3 so as to actuate the lock bolt. The cam member 6 has a first engagement portion (the end wall of recess 57 as shown in Fig. 6A). Also provided is an electrically-driven rotation blocking mechanism (solenoid 7, slider 8, and claw 68) for engaging the first engagement portion of the cam member when the lock bolt 1 is at the retreat position so as to block the cam member from rotating in a direction that would cause the lock bolt to move from the retreat position to the protruding position. The electrically-operated steering lock device further comprises a holding portion (the projection extending from the end wall of recess 57 as shown in Fig. 6A) for holding the rotation blocking mechanism in a state in which rotation of the cam member is blocked.

Because of the provision of this holding portion, if solenoid 7 attempts to pull in slider 8 (due to a malfunction for example) while the lock bolt 1 is in the retreat position, the slider is prevented from being disengaged from the cam member due to the engagement between the holding portion and claw 68 as shown in Fig. 10A. Accordingly, the cam member 6 is prevented from moving lock bolt 1 to its protruding position, whereby the steering shaft 97 is prevented from being locked during running of a vehicle. Claim 6 is representative of the steering lock device as described above.

Claims 1-5 were rejected under 35 U.S.C. § 102(b) as being anticipated by Suzuki et al. This rejection is respectfully traversed, and Suzuki et al. is not applicable with regard to the newly added claims for the following reasons.

Claim 6 recites an electrically-operated steering lock device that includes an electrically-driven rotation blocking mechanism for engaging said first engagement portion when said lock bolt is at said retreat position so as to block said cam member from rotating in a direction that would cause said lock bolt to move from said retreat position to said protruding position, and also includes a holding portion for holding said rotation blocking mechanism in a state in which rotation of said cam member is blocked.

In rejecting claim 1 as being anticipated by Suzuki et al., the Examiner took the position that Suzuki et al. discloses an electrically-operated steering lock device comprising a lock bolt 14 which is moveable between a protruding position where a steering shaft 9 is locked and a retreat position where the steering shaft is unlocked, a cam member 34 which is rotated by an electric motor 18 so as to actuate the lock bolt, an electrically-driven rotation blocking mechanism 43 which engages with a first engagement portion 33a of the cam member when the lock bolt is at the retreat position, and a holding portion 33b for holding the rotation blocking mechanism when rotation of the cam member is blocked.

It is respectfully submitted that Suzuki et al. does not disclose neither the "rotation blocking mechanism" nor the "holding portion" as recited in claim 6.

In this regard, in the steering lock apparatus of Suzuki et al. "43" corresponds to a gear mechanism that is composed of an output gear 19, a relay gear 41 and gear portion 28. This gear mechanism 43 transmits a driving force of motor 18 to cam member 20 and driven member 21, and in no way serves to block the cam member 20 from rotating in a direction for moving the lock bolt 14 from its retreat position to its protruding position. Thus, it is respectfully submitted that the gear mechanism 43 is not a rotation blocking mechanism as required by claim 6.

Additionally, step portion 33a of the driven body 21 engages driven body damper 37 for restraining rotation of the driven body 21 when the lock bolt 14 is moved from the protruding position to the retreat position, but this step portion is not a portion for blocking rotation of the cam member 20 in a direction for moving the lock member 14 from the retreat position to the protruding position. No portion of gear mechanism 43 engages with this step portion 33a.

Because Suzuki et al. does not disclose the rotation blocking mechanism as claimed, this reference also does not disclose the holding portion as claimed. With regard to step portion 33b,

which the Examiner has equated to the claimed holding portion as recited in claim 6, like step portion 33a this step portion does not engage the gear mechanism 43 in any manner, and is for engaging the damper 37 so as to retrain rotation of the driven body 21 when the lock bolt 14 is moved from the retreat position to the protruding position. Thus, step portion 33b does not correspond to the holding portion as claimed.

For the above reasons, claim 6 is not anticipated by Suzuki et al., whereby claims 6-25 are allowable.

In view of the above amendments and remarks, it is respectfully submitted that the present application is in condition for allowance and an early Notice of Allowance is earnestly solicited.

If after reviewing this Amendment, the Examiner believes that any issues remain which must be resolved before the application can be passed to issue, the Examiner is invited to contact the Applicant's undersigned representative by telephone to resolve such issues.

Respectfully submitted,

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# **Electrically-Operated Steering Lock Device**

### **BACKGROUND OF THE INVENTION**

[0001] The present invention relates to an electrically-operated steering lock device which is prevented from locking the <u>a</u> steering shaft even if an electrically-operated member has malfunctioned due to noise or the like.

[0002] Conventionally, an electrically-operated steering lock device, as disclosed in, for example, Japanese Patent Laid-Open Publication No. 2002-234419, is so designed that a lock bolt (lock pin) is driven by a plate cam coupled to an electric motor so that the lock bolt is protruded toward a steering shaft so as to be engaged with the steering shaft, and thus the steering shaft is locked. Further, with an engagement concave portion formed in the plate cam, a plunger of a solenoid is engaged with the engagement concave portion so that even if the electric motor malfunctions, the lock bolt is prevented from popping out by keeping the plate cam from rotating with a hold via being held by the plunger.

However, in this electrically-operated steering lock device, there has been a problem in that when the solenoid and the electric motor are simultaneously driven due to noise or the like during a-running of a vehicle, the solenoid may pull in the plunger so that its engagement with the plate cam is released, where wherein at this timing time of releasing, release the electric motor rotates to make cause the lock bolt protruded to protrude toward the steering shaft, thus locking the steering shaft.

#### SUMMARY OF THE INVENTION

[0004] The present invention having been accomplished in view of these and other problems of the prior art, an object of the invention is to provide an electrically-operated steering

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lock device which keeps-prevents the lock bolt from protruding even if electric currents flow through the electric motor and the solenoid at the same time by an arrangement that in which a protrusion blocking means device, such as a solenoid or the like, for blocking the protrusion of the lock bolt, is prevented from malfunctioning even if an electric current is passed through the protrusion blocking means device during a running of a vehicle.

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[0005] In order to achieve the above-described object, the present invention provides an electrically-operated steering lock device comprising a lock bolt which is movable between a protrusion-protruding position where a steering shaft is locked and a retreat position where the steering shaft is unlocked, and a cam member which is rotated by an electric motor to actuate the lock bolt, the. The electrically-operated steering lock device further comprising comprises: a rotation blocking means-mechanism which is electrically driven and which, when the lock bolt is placed at the retreat position, engages with a first engagement portion formed in the cam member to block rotation of the cam member; and a holding means-portion for holding the rotation blocking means-mechanism in a state that in which rotation of the cam member is blocked.

[0006] With this constitution, even if electric currents have flowed simultaneously through both the electric motor and the rotation blocking means-mechanism due to noise or the like, so that the cam member and the rotation blocking means-mechanism are driven, the rotation blocking means-mechanism is held by the holding means portion, so and thereby never activated. Thus, the a possibility that the lock bolt may erroneously be protruded to make lock the steering shaft locked can be reliably eliminated-reliably.

[0007] Further, in the electrically-operated steering lock device according to the invention, the cam member may act to move the lock bolt to the protrusion-protruding position when the electric motor is forwardly rotated-forward, and to move the lock bolt to the retreat position when the electric motor is reversely rotated-reverse.

[0008] With this constitution, the lock bolt is not actuated to the protrusion-protruding position by a biasing force of a spring or the like, but is moved to the protrusion-protruding position by the cam member. Therefore, even in the event that strong vibrations are caused to act during the running of the a vehicle with the rotation blocking means-mechanism and the cam member disengaged from each other, the lock bolt is never mis-protruded, so that the safety can be further improved.

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[0009] Further, in the electrically-operated steering lock device according to the invention, the holding means-portion may be a second engagement portion formed in the cam member, where wherein in a state that engagement with the second engagement portion has been released by reverse rotation of the electric motor, the lock bolt is protrudable-protruded by forward rotation of the electric motor.

[0010] With this constitution, in order to make-cause the lock bolt to be protruded, the electric motor needs to be rotated once rotated-in reverse before the electric motor is forwardly rotated-forward, and then, in this state, the engagement between the rotation blocking means mechanism and the holding means-portion needs to be released. There is no possibility that noise may enter at such a timing time, so that mis-operations of the lock bolt can be blocked with more reliability.

[0011] Furthermore, in the electrically-operated steering lock device according to the invention, a cam portion of the cam member may be formed in such a manner that the lock bolt is not actuated at a time of <u>a</u> release operation that the during which engagement with the second engagement portion is released by reverse rotation of the electric motor.

[0012] With this constitution, since there is no need for rotating the lock bolt from the retreat position further in the retreat direction, the a working range of the lock bolt can be narrowed, thus making it possible to downsize the electrically-operated steering lock device.

[0013] Also, the electrically-operated steering lock device according to the invention may further comprise <u>a</u> lock bolt holding <u>means-device</u> for holding the lock bolt at the retreat position while the lock bolt is placed at the retreat position.

[0014] With this constitution, even if the cam member is not engaged with the lock bolt, the lock bolt can be prevented from rattling, so that unusual noise due to vibrations of the lock bolt never occurs.

## **BRIEF DESCRIPTION OF THE DRAWINGS**

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[0015] The present invention will be further described with reference to the accompanying drawings wherein like reference numerals refer to like parts in the several views, and wherein:

[0016] Fig. 1 is a cross-sectional bottom view showing an electrically-operated steering lock device according to an embodiment of the invention;

[0017] Fig. 2 is a cross-sectional front view taken along the line A - A of Fig. 1;

[0018] Fig. 3 is a cross-sectional front view taken along the-line B - B of Fig. 1;

[0019] Fig. 4 is a cross-sectional front view showing an unlocked state of Fig. 2;

[0020] Fig. 5 is a front view of the <u>a</u> rotor of Fig. 2;

[0021] Fig. 6A is a front view showing the a cam member of Fig. 2, and Fig. 6B is a side view of Fig. 6A;

[0022] Fig. 7 is a front view showing a coupled state of the rotor and the cam member of Fig. 2;

[0023] Fig. 8A is a plan view showing the <u>a</u> slider of Fig. 3, Fig. 8B is a front view of Fig. 8A, and Fig. 8C is a side view of Fig. 8A;

[0024] Fig. 9A is a front view of a main part showing a state that in which the cam member of Fig. 3 has been rotated to the an unlocked side to the most its utmost, and Fig. 9B is a front view of a main part showing a state of the a switch cam shown in Fig. 1 under the state of Fig. 9A; and

[0025] Fig. 10A is a front view of a main part showing a state that the slider and the cam member of Fig. 3 have been engaged with each other, and Fig. 10B is a front view of a main part showing a state of the switch cam shown in Fig. 1 under the state of Fig. 10A.

## DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

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[0026] Figs. 1, 2 and 3 show an electrically-operated steering lock device of an embodiment according to the present invention. This electrically-operated steering lock device is so constructed that two lock bolts 1, 2, an electric motor 3, a final reduction gear 4, a rotor 5, a cam member 6, a solenoid 7, a slider 8, a switch cam 9, and a base 10 for placing thereon these members are all arranged within a housing 11 and housed with a cover 12.

As shown in Fig. 2, the-first lock bolt 1 comprises a platy-plate-like body portion 16, and on one side of a steering shaft 97 side of the body portion 16 is provided a protruding piece 17 that is upwardly protruding. Also, on the other another side of the steering shaft 97 side of the body portion 16 is provided a bearing hole 18. This first lock bolt 1 is pivotally held at its bearing hole 18 on a shaft portion 12a which is provided so as to protrude from the cover 12. As a result, the first lock bolt 1 is disposed right-hand downward downwardly of the steering shaft 97 in Fig. 2 so as to be rotatable about the shaft portion 12a. Then, when the first lock bolt 1 is rotated in a protruding direction (clockwise), the protruding piece 17 thereof is protruded outward-outwardly from an opening portion 12c of the cover 12. The-This protruded

protruding piece 17 is engaged with a receiving portion 98 of the steering shaft 97 of the <u>a</u> vehicle, thereby locking the steering shaft 97.

In the first lock bolt 1, a spring holding hole 20 for holding one end of an action spring 24, which is a lock bolt holding means member, is formed on a lower side of the body portion 16. This action spring 24, which is variable in its biasing direction depending on the a position of the first lock bolt 1, has the other another end held by the cover 12. The action spring 24 biases the first lock bolt 1 toward the protruding direction (clockwise) when the first lock bolt 1 is placed at a protrusion protruding position (a position shown in Fig. 2), and biases the first lock bolt 1 toward a retreat direction (counterclockwise) when the first lock bolt 1 is placed at a retreat position (a position shown in Fig. 4). Further, in the first lock bolt 1, a coupling-use concave portion 21 for actuating the later-described second lock bolt 2 is formed on a lower side of the protruding piece 17 of the body portion 16, while a generally arc-shaped contact surface 22 with which the later-described cam member 6 is to be brought into contact is formed on a lower side face of the body portion 16.

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The second lock bolt 2 comprises a generally U-shaped, platy-plate-like body portion 25, and on one side of a steering shaft 97 side of the body portion 25 is provided a protruding piece 26 that is upwardly protruding, while downward of this protruding piece 26 is provided a coupling convex portion 27 which is elongated into the coupling-use concave portion 21 of the first lock bolt 1 so as to be engaged therewith. Also, on the other another side of the steering shaft 97 side of the body portion 25 is bored a bearing hole 28. This second lock bolt 2 is pivotally held at its bearing hole 28 on a shaft portion 12b which is provided so as to protrude from the cover 12. Thus, the second lock bolt 2 is rotatably disposed at a position counter to the first lock bolt 1 with the steering shaft 97 interposed therebetween, so that the coupling convex portion 27 is engaged with the coupling-use concave portion 21 of the first lock bolt 1,

thereby making the second lock bolt 2 interlocked with the first lock bolt 1. Then, when the first lock bolt 1 is rotated in the protruding direction, the second lock bolt 2 is rotated in linkage together in the protruding direction, so that the protruding piece 26 of the second lock bolt 2 is protruded outward-outwardly from the opening portion 12c of the cover 12, simultaneously with the protruding piece 17 of the first lock bolt 1. The—This protruded protruding piece 26, together with the protruding piece 17 of the first lock bolt 1, is engaged with the receiving portion 98 of the steering shaft 97 of the vehicle, thereby locking the steering shaft 97.

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The electric motor 3, which is powered by unshown lines laid in the housing 11, is driven into forward-forwardly and reverse rotation reversely rotated by an internal control circuit 60. Also, the electric motor 3, as shown in Figs. 1 and 2, acts to rotate the final reduction gear 4 via a rotating mechanism 31. This rotating mechanism 31 is made up of a worm 33 fitted to a driving shaft of the electric motor 3, a large-diameter worm gear 34 to be meshed with the worm 33, and a coaxial small-diameter gear 36 provided integrally with a rotating shaft 35 of the worm gear 34. The small-diameter gear 36 is to be meshed with a segment gear 38 formed on the an outer periphery of the final reduction gear 4. As a result of this, when the electric motor 3 is rotated forward forwardly, the small-diameter gear 36 is rotated counterclockwise in Fig. 2. As the small-diameter gear 36 is rotated counterclockwise, the final reduction gear 4 is rotated clockwise. Conversely, as the electric motor 3 is reversely rotated reverse, the small-diameter gear 36 is rotated clockwise in Fig. 2, so that the final reduction gear 4 is rotated counterclockwise.

[0031] The final reduction gear 4, which is formed into a generally fan shape with the segment gear 38 formed on its outer periphery, is housed within a housing concave portion 12d formed in the cover 12 by a plate 84 that is a lid member so that the final reduction gear 4 can be rotated in the housing concave portion 12d. A generally D-shaped coupling hole 39 is formed

at a center of the final reduction gear 4, and a coupling portion 40 of the rotor 5, which is a coupling member, is fitted into this coupling hole 39 and thereby coupled to the final reduction gear 4. Then, this final reduction gear 4 and the rotor 5 are pivotally held by a shaft 45 extending through a shaft insertion hole 41 provided at a rotational center of the rotor 5 and partly formed into a generally D-shaped cross section. The shaft 45 has an-a\_tip end portion 46 having a generally D-shaped cross section, a rotational portion 47 having a circular-shaped cross section, a fitting portion 48 having a generally D-shaped cross section, and a rear end portion 49 having a circular-shaped cross section, as listed-shown from the upside-upper side toward the lower side in Fig. 1. In this shaft 45, the rotational portion 47 and the rear end portion 49, which are circular-shaped in cross section, are pivotally held on the base 10 and the plate 84, respectively, so that the shaft 45 itself is held to be rotatable. Further, the fitting portion 48 of the shaft 45 is kept-maintained fitted in the shaft insertion hole 41 of the rotor 5, so that as the final reduction gear 4 is rotated, the rotor 5 and the shaft 45 are rotated together. It is noted that the final reduction gear 4 is regulated in its rotational range by a stopper 12e protrusively provided on the cover 11.

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[0032] As shown in Fig. 5, on the an upper surface of the coupling portion 40 of the rotor 5 is formed a flange portion 42 that is radially protruding protrudes in a generally fan shape, while on the an upper surface of the flange portion 42, an upwardly-protruding spring-use engagement convex portion 43 and a rotation-use engagement convex portion 44 are provided at specified positions.

[0033] The cam member 6, as shown in Figs. 6A and 6B, is composed of a cam portion 52 formed so that the a distance from a bearing hole 51 formed at a rotational center is varied, a coupling convex portion 53 which is provided so as to protrude from the cam portion 52 toward the rotor 5 and which is to be engaged and coupled with the rotor 5, and a bushing portion 54 is

provided so as to cylindrically protrude from the cam portion 52 toward the rotor 5-side, where wherein a bearing hole runs through the-a center of the bushing portion 54. Then, this cam member 6, as shown in Fig. 7, is pivotably held by the rotational portion 47 of the shaft 45, and the coupling convex portion 53 of the cam member 6 is to be engaged with the rotation-use engagement convex portion 44 of the rotor 5 so that the cam member 6 is rotated by the-rotation of the rotor 5. Between the cam member 6 and the rotor 5<sub>3</sub> is provided a spring 14, which is disposed in a state such that one end of the spring 14 is engaged with the spring-use engagement convex portion 43 of the rotor 5 while the other another end is engaged with the coupling convex portion 53 of the cam member 6. That is, when the rotor 5 is rotated counterclockwise, the rotation-use engagement convex portion 44 is engaged with the coupling convex portion 53 so that the rotor 5 and the cam member 6 are rotated together. On the other hand, as the rotor 5 is rotated clockwise, the rotor 5 causes the cam member 6 to be rotated via the spring 14. As a result of this, even when the cam member 6 is unrotatable not rotated, the rotor 5 is rotated clockwise so that biasing force can be accumulated on-in the spring 14.

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The cam portion 52 of the cam member 6 has a first cam wall surface 55 which presses the a contact surface 22 of the first lock bolt 1 toward the steering shaft 97 by via the cam member 6 rotating clockwise so that the first lock bolt 1 is rotated from the retreat position to the protrusion protruding position. The cam member 6 also has a second cam wall surface 56 which presses the contact surface 22 of the first lock bolt 1 outward outwardly (toward a side away from the steering shaft 97) by via the cam member 6 rotating counterclockwise so that the first lock bolt 1 is rotated from the protrusion protruding position to the retreat position. Further, on the a peripheral surface of the cam portion 52 is formed a receiving recess 57 for receiving the slider 8 coupled to the later-described solenoid 7. This receiving recess 57 is provided with an engagement portion 58 as a holding means portion which is to be engaged with

the slider 8 to restrict the slider 8 from moving toward the solenoid 7. The engagement portion 58 comprises an end wall of the receiving recess 57 which is the <u>a</u> first engagement portion, and a claw-like projection which is the <u>a</u> second engagement portion.

[0035] The solenoid 7, which is fixed on the cover 12, is actuated by a start-up signal from the internal control circuit 60, and a plunger 61 is moved toward the solenoid 7 side in its actuated state. To a tip end portion of this plunger 61 is coupled the slider 8 which is to be engaged with the end wall of the receiving recess 57 of the cam member 6 to block the rotation of the cam member 6. Between this slider 8 and the solenoid 7 is provided a spring 62 with the plunger 61 serving as the an axial center, so that the slider 8 is biased toward the cam member 6 by the a biasing force of the spring 62 while the solenoid 7 is not actuated. In this embodiment, the solenoid 7, the slider 8 and the spring 62 constitute a rotation blocking means mechanism.

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The slider 8, as shown in Figs. 8A, 8B and 8C, comprises a generally rectangular-parallelopiped body portion 65, where—wherein a coupling recess 66 is provided for fitting thereinto a tip end of the plunger 61 of the solenoid 7 so that the plunger 61 is coupled therewith, and an actuation detection piece 67 provided so as to protrude from the body portion 65 is formed on a downward side of the coupling recess 66. This actuation detection piece 67 serves to press a detection portion 81 of a solenoid switch 80 (see Fig. 3) for detecting a move movement of the slider 8, and this solenoid switch 80 is turned on when the slider 8 has moved to a position where the slider 8 is engageable engaged with the cam member 6. Also, on the a right-side frontage of the body portion 65 is protrusively provided an engagement claw portion 68 which intrudes into the receiving recess 57 of the cam member 6 so as to be engaged with the engagement portion 58. Further, on rear-side upper and lower surfaces of the body portion 65 are provided two rail recesses 69, respectively, into which rail-shaped convex portions 12f (one of which is shown in Fig. 1) of the cover 12 are positioned to restrict moves—movement of the

slider 8, except its movement in rightward and leftward moves in directions of Fig. 3. This is intended to prevent the slider 8 from moving upward-upwardly and downward-downwardly in Fig. 3 even if the a rotational force of the cam member 6 acts on the slider 8, so that forced thrust is not applied to the plunger 61 of the solenoid 7, thus producing an effect of preventing failures of the solenoid 7.

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The switch cam 9, which is purposed whose purpose is to detect the a rotational position of the final reduction gear 4, is coupled to the tip end portion 46 of the shaft 45 as shown in Fig. 1 so as to be rotated together with the shaft 45. On the an outer circumferential wall of this switch cam 9 is provided a switch pressing portion 72 which protrudes toward the an outer side in a fan shape as shown in Fig. 9B. This switch pressing portion 72 presses a detection portion 75 of an unlock switch 74 for detecting a rotation of the switch cam 9, and this unlock switch 74 is turned on when the shaft 45, the rotor 5 and the cam member 6, all of which are rotated together counterclockwise, have been rotated to a specified position shown in Fig. 9A.

[0038] Next, operation of the electrically-operated steering lock device having the above constitution is described.

Fig. 2 and Fig. 3 show a state that-in which the lock bolts 1, 2 of the electrically-operated steering lock device are in a protrusion-protruding position, wherein where they are protruded from the opening portion 12c of the cover 12 in a stopped automobile. In this state, the electric motor 3 is in a halt state and the final reduction gear 4 is at a clockwise-rotated position, where one end face of the final reduction gear 4 is in contact with the stopper 12e. Also, the solenoid 7 is in an OFF state, and the slider 8 is kept in contact with a side face of the cam member 6 by the a biasing force of the spring 62. At this time, the plunger 61 is in the a pushed-out position, and the solenoid switch 80, operative in response to the actuation detection

piece 67 of the slider 8, is in the OFF state. Further, the unlock switch 74 for detecting a rotation of the switch cam 9 is in the OFF state.

Upon input of a signal for releasing the lock lockage of steering from a vehicle-side control circuit (not shown) in this state, a signal for reversely rotating reverse the electric motor 3 is delivered from the internal control circuit 60, and with a current fed to the electric motor 3, the electric motor 3 is rotated, by which the final reduction gear 4, the rotor 5, the shaft 45, the switch cam 9, and the cam member 6, of which the coupling convex portion 53 is engaged with the rotation-use engagement convex portion 44 of the rotor 5, are rotated together counterclockwise via the rotating mechanism 31.

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of the cam member 6 is rotated counterclockwise, the second cam wall surface 56 of the cam member 6 is brought into contact with the contact surface 22 of the first lock bolt 1, and the first lock bolt 1 is pressed in the retreat direction, thereby rotated—rotating counterclockwise. Also, the coupling convex portion 27 of the second lock bolt 2 protruded into the coupling-use concave portion 21 of the first lock bolt 1 is moved while keeping the maintaining engagement, and thus the second lock bolt 2 is also pressed in the retreat direction, thereby rotated rotating clockwise. Then, as shown in Fig. 4, the protruding pieces 17, 26 of the first and second lock bolts 1, 2 separate move away from the receiving portion 98 of the steering shaft 97 of the vehicle. As a result, the engagement between the steering shaft 97 and the lock bolts 1, 2 is released, by which the steering shaft 97 is unlocked.

Then, when the cam member 6 is rotated up to a position shown in Fig. 9A, i.e., a position where the other end face of the final reduction gear 4 makes contact with the stopper 12e, the engagement claw portion 68 of the slider 8 goes beyond the engagement portion 58 from on the a side face of the cam member 6, thereby plunging into the receiving recess 57 from the

position of Fig. 3 by the-a biasing force of the spring 62. In this case, the solenoid switch 80 comes to an ON state, and the unlock switch 74 also comes to an ON state.

The internal control circuit 60, upon reception of signals indicating an ON for the solenoid switch 80 and an ON for the unlock switch 74, outputs a signal to rotate the electric motor 3 a-little slightly forward. As a result, with a current fed to the electric motor 3, the electric motor 3 is rotated a-little slightly forward, so that the final reduction gear 4 is rotated a little slightly clockwise via the rotating mechanism 31, where wherein the rotor 5, the shaft 45, and the switch cam 9, together with the cam member 6, that is pressed against the rotor 5 via the spring 14, are rotated a-little slightly clockwise. In this case, even if the cam member 6 is rotated a-little slightly clockwise, the first cam wall surface 55 of the cam member 6 is not engaged with the contact surface 22 of the first lock bolt 1, and the first lock bolt 1, which is biased toward the retreat position by the action spring 24, keeps-is maintained held in the retreat position.

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Then, as shown in Figs. 10A and 10B, the engagement claw portion 68 of the slider 8 is engaged with the engagement portion 58 while kept in contact with the a bottom face of the receiving recess 57. In this case, the unlock switch 74 is turned OFF, by which the internal control circuit 60, receiving a signal indicating an OFF of the unlock switch 74, cuts off the power supply to the electric motor 3.

In this state, even if an action for rotating the cam member 6 clockwise is exerted in the event that, for example, noise has entered into the internal control circuit 60, from which a signal for <u>forwardly</u> rotating <u>forward</u> the electric motor 3 is issued so that the electric motor 3 is rotated <u>forward</u> forward, the cam member 6 never rotates <u>by via</u> its clockwise rotation being restricted by the slider 8, of which the engagement claw portion 68 is engaged with the end wall, as a part of the engagement portion 58, of the receiving recess 57 of the cam member 6. Thus,

the lock bolts 1, 2 are never moved from the retreat position to the protrusion protruding position, so that the steering shaft 97 is never locked.

[0046] Also, even in the event that special noise has entered into the internal control circuit 60, from which a signal for <u>forwardly</u> rotating <u>forward</u> the electric motor 3 and a signal for activating the solenoid 7 are issued simultaneously, the plunger 61, even if subject to an attempt for attracting the plunger 61, never moves by virtue of the engagement of the engagement claw portion 68 of the slider 8 with the claw-like projection as a part of engagement portion 58 of the cam member 6. Further, since the cam member 6 is subject to an action of clockwise rotational force by the electric motor 3, the <u>an</u> engaging force between the engagement claw portion 68 of the slider 8 and the engagement portion 58 of the cam member 6 becomes larger proportionally, <u>thereby</u> allowing the plunger 61 to be held securely. Thus, the cam member 6 is never allowed to rotate.

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In the event that a signal for <u>reversely</u> rotating <u>reverse</u>-the electric motor 3 and a signal for activating the solenoid 7 have been issued simultaneously from the internal control circuit 60 on account of an abnormality signal, the plunger 61 is attracted, but the cam member 6 has been rotated counterclockwise and therefore the lock bolts 1, 2 never protrude. Then, upon a cease of the abnormality signal, the solenoid 7 is stopped from activation, the engagement claw portion 68 of the slider 8 is protruded into the receiving recess 57 by <u>the-a\_biasing</u> force of the spring 62, thus becoming <u>engageable-engaged</u> with the engagement portion 58 while kept in contact with the bottom face of the receiving recess 57.

[0048] As described above, even if noise has entered into the internal control circuit 60 during the running of the vehicle, so that a signal for forwardly rotating forward—the electric motor 3 and a signal for driving the solenoid 7 are issued from the internal control circuit 60, the lock bolts 1, 2 are never moved in the protruding direction, hence high safety is realized.

Further, in the electrically-operated steering lock device of this embodiment, not that the lock bolts 1, 2 are not actuated to the protrusion-protruding position by biasing force of a spring or the like, but that the lock bolts 1, 2 are moved to the protrusion-protruding position by the cam member 6. Therefore, even in the event that the engagement between the engagement claw portion 68 of the slider 8 and the engagement portion 58 of the cam member 6 is released for some reason, thereby causing strong vibrations to act during the running of the vehicle, the lock bolts 1, 2 are never mis-protruded, so that the safety can be further improved.

[0050] When a signal for locking the-steering is transmitted from the <u>a</u> vehicle-side control circuit to the internal control circuit 60, a signal for <u>reversely</u> rotating <u>reverse</u>-the electric motor 3 is issued from the internal control circuit 60, in which case the cam member 6 is rotated from the position shown in Fig. 10A to the position shown in Fig. 9A. Then, the unlock switch 74 is turned ON, and a signal for activating the solenoid 7 is issued from the internal control circuit 60, <u>thereby</u> causing the solenoid 7 to be driven, so that the plunger 61 and the slider 8 are attracted to positions where <u>those these members</u> escape from the receiving recess 57 against the <u>a</u> biasing force of the spring 62.

[0051] In this case, even if the cam member 6 is rotated counterclockwise to the position shown in Fig. 9A, the second cam wall surface 56 of the cam member 6 is not engaged with the contact surface 22 of the first lock bolt 1, so that the first lock bolt 1 is not activated.

Then, when the solenoid switch 80 is turned off, a signal for <u>forwardly</u> rotating forward-the electric motor 3 is issued from the internal control circuit 60. Thus, the electric motor 3 is <u>forwardly</u> rotated-forward, so that the final reduction gear 4, the rotor 5, the shaft 45 and the switch cam 9, together with the cam member 6, pressed against the rotor 5 via the spring 14, are rotated clockwise via the rotating mechanism 31.

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As the cam member 6 is rotated, the first cam wall surface 55 of the cam member 6 is brought into contact with the contact surface 22 of the first lock bolt 1, and the first lock bolt 1 is pressed in the protruding direction so as to be rotated clockwise. Further, the coupling convex portion 27 of the second lock bolt 2 protruding into the coupling-use concave portion 21 of the first lock bolt 1 is moved in-into engagement, and thus the second lock bolt 2 is also pressed in the protruding direction so as to be rotated counterclockwise. In-During this operation, the protruding pieces 17, 26 of the first and second lock bolts 1, 2 are protruded outward-outwardly from the opening portion 12c of the cover 12, thereby entering into the receiving portion 98 of the steering shaft 97 of the vehicle, and thereby being engaged with the receiving portion 98 of the steering shaft 97, by which the steering shaft 97 is locked.

Then, as the cam member 6 is rotated to the position shown in Fig. 2, one end face of the final reduction gear 4 is brought into contact with the a side wall of the stopper 12e of the cover 12, thereby being inhibited from rotation, and after a specified time elapse, the power supply to the motor 3 is cut off. Thereafter, the internal control circuit 60 halts the conduction to the solenoid 7, where wherein the slider 8 comes into contact with the side face of the cam member 6 by the a biasing force of the spring 62. Thus, the electrically-operated steering lock device comes into the a lock state as shown in Figs. 2 and 3.

As described above, with the a constitution that the lock bolts 1, 2 are not actuated at the a time of the a release operation, during which that the engagement between the engagement claw portion 68 of the slider 8 and the engagement portion 58 of the cam member 6 is released by reversely rotating reverse the electric motor 3, there is no need for rotating the lock bolts 1, 2 from the retreat position further in the retreat direction, so that the a working range of the lock bolts 1, 2 can be narrowed, thus making it possible to downsize the electrically-operated steering lock device.

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[0056] It is noted that this embodiment may be changed in the following modes.

Although the solenoid 7 is driven after the first lock bolt 1 is moved from the position shown in Fig. 10A to the position shown in Fig. 9A in the above embodiment, the solenoid 7 may also be driven simultaneously with movement of lock bolt 1. In this case, because of the concurrent timing of conduction for both the solenoid 7 and the electric motor 3, the a response speed can be made faster increased to a degree of its effect.

[0058] Although the solenoid 7, the slider 8 and the spring 62 constitute a rotation blocking means-mechanism in the above embodiment, it is also possible that the tip end portion of the plunger 61 of the solenoid 7 is engaged directly with the engagement portion 58 of the cam member 6, or that the slider 8 is activated by an electric motor or the like.

[0059] Although the present invention has been fully described by way of examples with reference to the accompanying drawings, it is to be noted that various changes and modifications will be apparent to those skilled in the art. Therefore, unless otherwise—such changes and modifications depart from the scope of the present invention, they should be construed as being included therein.

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## ABSTRACT OF THE DISCLOSURE

An electrically-operated steering lock device, having-includes a lock bolt which is movable between a protrusion-protruding position where the a steering shaft is locked and a retreat position where the steering shaft is unlocked, and a cam member which is rotated by an electric motor so as to actuate the lock bolt, is further provided with. The electrically-operated steering lock device also includes: a rotation blocking means-mechanism which is electrically driven and which, when the lock bolt is placed at the retreat position, engages with a first engagement portion formed in the cam member to block rotation of the cam member, and a holding means-portion for holding the rotation blocking means-mechanism in a state that rotation of the cam member is blocked. In the electrically-operated steering lock device, the steering shaft is not locked during a running of a vehicle even if any electrically-operated member has malfunctioned due to noise or the like.

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